



### Two-Variable State Space Model of the Combination

Applying the author's idea of Ref. 11, following the block diagrams of Figs. 2 and 3a, results in a mathematical model of each block, which can then be combined and transferred into one integrated block as shown in Fig. 3d.

#### Inlet Frequency Domain Model

Applying the method of Ref. 11, the transfer function of the inlet exit total pressure increment  $\Delta P_2$  from the bypass airflow increment  $\Delta W_{BP}$  is

$$G_{IN,1} = \frac{\Delta P_2}{\Delta W_{BP}} = \frac{b_{61}S^5 + b_{51}S^4 + b_{41}S^3 + b_{31}S^2 + b_{21}S + b_{11}}{S^6 + a_{61}S^5 + a_{51}S^4 + a_{41}S^3 + a_{31}S^2 + a_{21}S + a_{11}} \quad (1)$$

and the transfer function of inlet shock position increment  $\Delta X_s$  from inlet exit airflow rate increment  $\Delta W_{IN}$  is

$$G_{IN,2} = \frac{\Delta X_s}{\Delta W_{IN}} = \frac{b_{12}}{S^6 + a_{62}S^5 + a_{52}S^4 + a_{42}S^3 + a_{32}S^2 + a_{22}S + a_{12}} \quad (2)$$

Equations (1) and (2) represent the dynamic responses of the downward and upward wave propagation of the flow disturbance, respectively. Because the denominators of the two inlet transfer functions  $G_{IN,1}$  and  $G_{IN,2}$  are different for the cross-sectional areas of the upward convergent portion  $X_5$ - $A_{BP}$  and the downward extension duct portions 0-2 (see Fig. 1) are different, their volume dynamics are different. Each of them is a sixth-order model of part of the inlet. Their logarithmic amplitude vs frequency and phase vs frequency characteristics are different, as shown in Fig. 4. The whole inlet is of twelfth order and is a medium frequency (e.g., volume dynamics) model with a time constant of 0.01–0.05 s. The resultant upward and downward wave propagation effects have been included in a single sixth-order characteristic equation<sup>5,12</sup> for studying the inlet dynamics and its control only. Surely, the sixth-order transfer function  $G_{IN,1}$  or  $G_{IN,2}$  can be reduced to, say, third order by further approximation. But the accuracy will be greatly decreased.

#### Engine State Equation and Frequency Domain Model

A real engine covers a wide range of frequencies.<sup>13</sup> The rotor inertia and thermal lag of turbine blades represent low-frequency dynamics of the engine with a time constant of 0.1–1 s. The gas compressibilities of the compressor and turbine cascade volumes represent high-frequency dynamics of the engine with a time constant of 0.00001–0.001 s. Therefore, for simplicity, the high-frequency volume dynamics of the engine are ignored. The gas compressibilities of the combustion chamber and exhaust nozzle volumes and combustion delay also represent medium-frequency dynamics of the engine with a time constant of 0.01–0.03 s, but they are to be considered as a portion of the extension duct ( $L_T$  in Fig. 1) of the inlet for simplicity. Otherwise, the engine model will be very complex.

Under such assumptions, the engine state equation (with a control variable  $W_F$  and an intake condition  $P_2$ ) can be deduced from the nonlinear dynamic model developed by using Seldner's generalized simulation techniques,<sup>14</sup>

$$\begin{bmatrix} \dot{\Delta N} \\ \dot{\Delta T}_M \end{bmatrix} = \begin{bmatrix} a'_{11} & a'_{12} \\ a'_{21} & a'_{22} \end{bmatrix} \begin{bmatrix} \Delta N \\ \Delta T_M \end{bmatrix} + \begin{bmatrix} b'_{11} \\ b'_{21} \end{bmatrix} \Delta W_F + \begin{bmatrix} b'_{12} \\ b'_{22} \end{bmatrix} \Delta P_2 \quad (3)$$

Eliminating terms  $\dot{T}_M$  and  $T_M$  from Eq. (3) and then transforming them into transfer functions by Laplace transformation, we get

$$G_{EN,1} = \frac{\Delta N_1}{\Delta P_2} = \frac{d_{23}S + d_{13}}{S^2 + a_{24}S + a_{14}} \\ G_{EN,2} = \frac{\Delta N_2}{\Delta W_F} = \frac{b_{24}S + b_{14}}{S^2 + a_{24}S + a_{14}} \quad (4)$$

where

$$a_{14} = a'_{11}a'_{22} - a'_{12}a'_{21}, \quad b_{24} = b'_{11} \\ a_{24} = -a'_{11} - a'_{22}, \quad d_{13} = a'_{12}b'_{22} - a'_{22}b'_{21} \\ b_{14} = a'_{12}b'_{12} - a'_{22}b'_{11}, \quad d_{23} = b'_{21}$$

and  $\Delta N_1$  and  $\Delta N_2$  are deviations in the engine speed due to the change of inlet exit pressure  $P_2$  and fuel flow rate  $W_F$ .

#### Noise Model

The transfer function of the disturbance noise<sup>5</sup> is

$$G_{NS} = \frac{\Delta W_{NS}}{\Delta W_w} = \frac{d_{22}S + d_{12}}{S^2 + C_{22}S + C_{12}} \quad (5)$$

where  $C_{12} = \alpha_1\alpha_3$ ,  $C_{22} = \alpha_1 + \alpha_3$ ,  $d_{12} = \alpha_1\alpha_3$ , and  $d_{22} = \alpha_1\alpha_3/\alpha_2$ , in which  $\alpha_1$ ,  $\alpha_2$ , and  $\alpha_3$  are noise coefficients.

#### Frequency Domain Model of Bypass Door and Actuator

From Ref. 5 the transfer function of the bypass airflow increment  $\Delta W_c$  from the control increment  $\Delta U_{BP}$  of the actuator is

$$G_{BP} = \frac{\Delta W_c}{\Delta U_{BP}} = \frac{d_{11}}{S^2 + C_{21}S + C_{11}} \quad (6)$$

#### Fuel Pump

The transfer function of the fuel pump is

$$\Delta W_F = K_F \Delta U_F \quad (7)$$

where  $K_F$  is the pump gain.

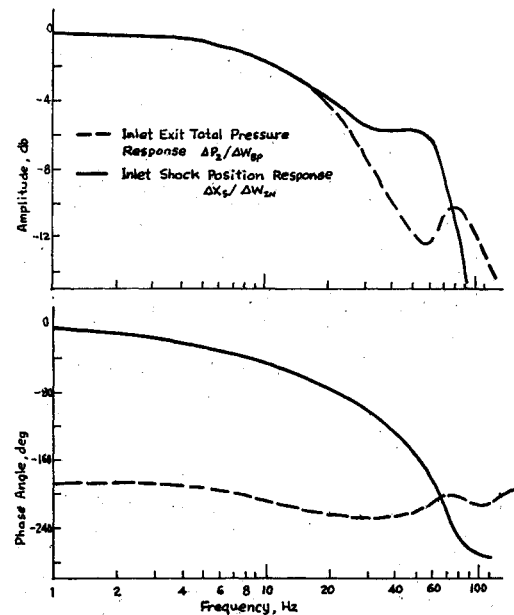


Fig. 4 Inlet frequency responses.

### Formulation of Model of the Inlet/Engine Combination

By the law of continuity, the inlet airflow must be equal to the sum of the bypass airflow plus the engine airflow at bypass section 0 of Fig. 1 at any time, or

$$W_{IN} = W_{BP} + W_{EN} \quad (8)$$

During transient periods,

$$\Delta W_{IN} = \Delta W_{BP} + \Delta W_{EN} \quad (9)$$

This equation is the basis of flow matching between the inlet and the engine. As the bypass doors are opening, a signal of inlet exit (or engine intake) pressure increment  $\Delta P_2$  is transmitted to the engine as shown in Fig. 3. This signal induces a change of the engine speed  $\Delta N$ . This is a forward-interacting signal of the inlet to the engine.

Similarly, as the engine speed changes, a signal of engine flow increment  $\Delta W_{EN}$  is fed back to the inlet directly. This is a backward-interacting signal of the engine to the inlet as shown in Fig. 3.

Now, the block diagram of the inlet/engine combination as a control object (the block bounded by dotted lines in Fig. 2) can be transformed into Fig. 3a by substituting the transfer functions of Eqs. (1-6) into each corresponding block.

For small disturbance, the engine flow is proportional to the engine speed, or

$$\Delta W_{EN} = -K_N \Delta N \quad (10)$$

where  $K_N$  is the gain of the engine flow to the engine speed.

Then, the block diagram of Fig. 3a can be rearranged as Fig. 3b. However, the interacting transfer function  $G_{IN,1}$ ,  $G_{EN,1}$ , and  $K_N$  of the two-input, and two-output inlet/engine combination are shown clearly.

The block diagram of Fig. 3 can be further simplified and transformed into Fig. 3c as shown. Block I is composed of the noise  $G_{NS}$  and the actuator  $G_{BP}$ , block II of the portion  $G_{IN,1}$  of the inlet and the engine, and block III of the other portion  $G_{IN,2}$  of the inlet. The state space model of the integrated combination is formulated by superposition of blocks I, II, and III as shown in Fig. 3d. Thus,

$$\dot{X} = AX + BU + G\Delta W_w \quad (11a)$$

$$R = CX + V \quad (11b)$$

where

$$X = [X_1 \ X_2 \ \cdots \ X_{17} \ X_{18}]^T, 18 \times 1 \text{ state vector}$$

$$U = [\Delta U_{BP} \ \Delta U_F]^T, 2 \times 1 \text{ control vector}$$

$$R = [\Delta R_{XS} \ \Delta R_N]^T, 2 \times 1 \text{ output vector}$$

$$V = [\Delta V_{XS} \ \Delta V_N]^T, 2 \times 1 \text{ measurement noise vector}$$

$$\Delta W_w = \text{stochastic airflow disturbance of the inlet}$$

$$\Delta R_{XS} = \Delta X_S + \Delta V_{XS}$$

$$\Delta R_N = \Delta N + \Delta V_N$$

$$A = \begin{bmatrix} A_1 & 0 & 0 \\ B_{12}C_1 & A_2 & 0 \\ B_3C_1 & -B_3C_{12} & A_3 \end{bmatrix}$$

$$B = \begin{bmatrix} B_{21} & 0 \\ 0 & B_{22} \\ 0 & 0 \end{bmatrix}$$

$$C = \begin{bmatrix} 0 & 0 & C_3 \\ 0 & C_{22} & 0 \end{bmatrix}$$

$$G = [B_{11} \ 0 \ 0]^T$$

The detail expression of the Jacobian matrices of **A**, **B**, **C**, and **G** are shown in Table 1. The derivation of Eq. (11) is shown in Appendix A.

### Solution of the State Space Model of the Combination

Since Eq. (11) is a set of stiff differential equations with a degree of stiffness around  $10^2$ , the dynamic behavior of the combination is solved by combining the Runge-Kutta method and the recursive method of eigenvalue-eigenvector (Appendix B) with different time steps to reach a reasonable computing time. The low-frequency engine model is solved by the former method with a time step of 0.01 s, while the medium-frequency inlet model is solved by the latter method with a time step of 0.001 s (less than 1/10 of the inlet time constant) in order to assure the stability of computation.<sup>15</sup>

### Optimal Control

#### Linear Quadratic Stochastic Optimal Control Problem

The linear time-invariant system described in the state-space expression of Eq. (11) with zero-mean, Gaussian noise vectors  $G\Delta W_w$  and **V** and their covariances are given by

$$E\{G\Delta W_w(t)[G\Delta W_w(t+\tau)^T]\} = Q_e\delta(\tau) \quad (12a)$$

$$E[V(t)V^T(t+\tau)] = Z_e\delta(\tau) \quad (12b)$$

$$E[G\Delta W_w(t)V^T(t+\tau)] = 0 \quad (12c)$$

$$E[G\Delta W_w(t)] = E[V(t)] = 0 \quad (12d)$$

where

$$Q_e = \begin{bmatrix} \text{PSD}(\Delta W_w) & 0 \\ 0 & 0 \end{bmatrix}_{18 \times 18} \quad (13a)$$

$$Z_e = \begin{bmatrix} \text{PSD}(\Delta V_{XS}) & 0 \\ 0 & \text{PSD}(\Delta V_N) \end{bmatrix}_{2 \times 2} \quad (13b)$$

and  $\text{PSD}(\Delta W_w)$ ,  $\text{PSD}(\Delta V_{XS})$ , and  $\text{PSD}(\Delta V_N)$  are the power spectral densities of  $\Delta W_w$ ,  $\Delta V_{XS}$ , and  $\Delta V_N$ , respectively.

Thus, the problem of linear quadratic stochastic optimal control is to select an optimal control vector  $U(t)$  to minimize the following quadratic performance index:

$$J = E \left\{ \int_0^\infty \frac{1}{2} [X^T(\tau)QX(\tau) + U^T(\tau)ZU(\tau)] d\tau \right\} \quad (14)$$

where **Q** is a positive semidefinite weighting matrix and **Z** a positive definite weighting matrix.

#### Formulation of Performance Index

For inlet control, the normal shock must be maintained downstream of the geometrical throat. Hence, the variance of the shock position increment  $E[(\Delta X_S)^2]$  is selected as one term of the performance index. For engine speed control, the increment of speed under any disturbances should be as small as possible. Therefore, the variance of the increment of engine speed  $E[(\Delta N)^2]$  is also selected as another term of the performance index. Furthermore, it is hoped to fulfil the control task with a minimum control effort, so that the variance of the increment of the bypass door control voltage  $E[(\Delta U_{BP})^2]$  and the variance of the increment of fuel flow  $E[(\Delta U_F)^2]$  are selected as the third and fourth terms of the performance index.

Hence, the performance index can be written as

$$\begin{aligned} J &= E \left\{ \frac{1}{2} \int_0^\infty [K_1 \Delta X_S^2 + K_2 \Delta N^2 + K_3 \Delta U_{BP}^2 + K_4 \Delta U_F^2] dt \right\} \\ &= E \left\{ \frac{1}{2} \int_0^\infty [X^T Q X + U^T Z U] dt \right\} \end{aligned} \quad (15)$$

Table 1 Elements of matrices A, B, C, G

<b>A =</b>																			
	$-c_{22}$	$-c_{12}$																	
	1																		
			$-c_{21}$	$-c_{12}$															
		1																	
	$d_{22}$	$d_{12}$			$d_{11}$	$-\mu_{65}$	$-\mu_{55}$	$-\mu_{45}$	$-\mu_{35}$	$-\mu_{25}$	$-\mu_{15}$								
						1													
							1												
								1											
									1										
										$-\mu_{26}$	$-\mu_{16}$								
										1									
	$d_{22}$	$d_{12}$			$d_{11}$	$-\mu_{51}$	$-\mu_{41}$	$-\mu_{31}$	$-\mu_{21}$	$-\mu_{11}$	$-\mu_{22}$	$-\mu_{12}$	$-a_{62}$	$-a_{52}$	$-a_{42}$	$-a_{32}$	$-a_{22}$	$-a_{12}$	
													1						
														1					
															1				
																1			
																	1		
																		1	
<b>B<sup>T</sup> =</b>	1																		
									1										
<b>G<sup>T</sup> =</b>	1																		
<b>C =</b>																			$b_{12}$
						$\mu_{53}$	$\mu_{43}$	$\mu_{33}$	$\mu_{23}$	$\mu_{13}$	$\mu_{24}$	$\mu_{14}$							

where

$$\mathbf{Q} = \mathbf{C}^T \begin{bmatrix} K_1 & 0 \\ 0 & K_2 \end{bmatrix} \mathbf{C} \quad \text{and} \quad \mathbf{Z} = \begin{bmatrix} K_3 & 0 \\ 0 & K_4 \end{bmatrix}$$

Factors  $K_1$  and  $K_2$  are used to regulate the magnitudes of the two controlled variables and  $K_3$  and  $K_4$  to limit the magnitudes of the two control variables. Every set of  $K_3$  and  $K_4$  can be used to minimize  $J$ , but a set composed of different magnitudes of  $\Delta U_{BP}$  and  $\Delta U_F$  are required in order that the controller can be designed without control saturation by a suitable choice of  $K_3$  and  $K_4$  if the maximum range in the variation of the two control variables are given.

#### Optimal Controller Design

In solving the control and estimation problem<sup>16</sup> for a quadratic performance index by the law of separation, the optimal estimate of the state vector and optimal control law can at first be treated separately and then combined into one optimal controller.

The optimal state estimate of the state vector can be generated by a Kalman filter of a continuous system described by

$$\dot{\hat{\mathbf{X}}} = \mathbf{A}\hat{\mathbf{X}} + \mathbf{B}\mathbf{U} + \mathbf{K}_e(\mathbf{R} - \mathbf{C}\hat{\mathbf{X}}) \quad (16)$$

where

$$\mathbf{K}_e = \mathbf{S}_e \mathbf{C}^T \mathbf{Z}_e^{-1} \quad (17)$$

is the optimal gain matrix and the variance matrix of estimation errors is the symmetric positive definite solution of the following algebraic Riccati matrix equation:

$$\mathbf{A}\mathbf{S}_e + \mathbf{S}_e \mathbf{A}^T - \mathbf{S}_e \mathbf{C}^T \mathbf{Z}_e^{-1} \mathbf{C} \mathbf{S}_e + \mathbf{Q}_e = 0 \quad (18)$$

For a linear quadratic output regulator problem, the optimal feedback control law is designed as

$$\mathbf{U} = -\mathbf{Z}^{-1} \mathbf{B}^T \mathbf{S}_e \mathbf{X} \quad (19)$$

where  $\mathbf{S}_e$  is the symmetric position definite solution of the algebraic Riccati matrix equation

$$\mathbf{S}_e \mathbf{A} + \mathbf{A}^T \mathbf{S}_e - \mathbf{S}_e \mathbf{B} \mathbf{Z}^{-1} \mathbf{B}^T \mathbf{S}_e + \mathbf{Q} = 0 \quad (20)$$

Substituting the optimal state estimate  $\hat{\mathbf{X}}$  from Eq. (13) for the state vector  $\mathbf{X}$  of Eq. (19), we have

$$\mathbf{U} = -\mathbf{Z}^{-1} \mathbf{B}^T \mathbf{S}_e \hat{\mathbf{X}} = -\mathbf{K}_e \hat{\mathbf{X}} \quad (21)$$

where

$$\mathbf{K}_e = \mathbf{Z}^{-1} \mathbf{B}^T \mathbf{S}_e \quad (22)$$

Substituting Eq. (21) into Eq. (16) and rearranging, we have the optimal controller equation

$$\dot{\hat{\mathbf{X}}} = (\mathbf{A} - \mathbf{B}\mathbf{K}_e - \mathbf{K}_e \mathbf{C})\hat{\mathbf{X}} + \mathbf{K}_e \mathbf{R} \quad (23a)$$

$$\mathbf{U} = -\mathbf{K}_e \hat{\mathbf{X}} \quad (23b)$$

The block diagram of the optimal inlet/engine combination control system is shown in Fig. 5.

#### Solution of Equations

Equation (23) can be solved by the eigenvalue-eigenvector method for a time-invariant state equation with the following recursive formulas:

$$\mathbf{Y}(t_{i+1}) = \mathbf{C}' \Phi(t_{i+1}) \quad (i = 0, 1, \dots, n) \quad (24a)$$

**Fig. 6 Transient responses of parameters of the integrated control system: a) downstream airflow disturbance  $\Delta W_w$ ; b) engine airflow  $\Delta W_{EN}$ ; c) shock position  $\Delta X_s$ ; d) engine speed  $\Delta N$ ; e) bypass door control  $\Delta U_{BP}$ ; f) fuel flow rate  $\Delta W_f$ ; g) compressor discharge pressure  $\Delta P_3$ ; h) turbine inlet gas temperature  $\Delta T_4$ .**

$$\mathbf{X}_1 = [X_1 \quad X_2 \quad X_3 \quad X_4]^T \quad (\text{A5a})$$

$X_1, X_2, X_3, X_4$  = state variables of bypass door and actuator noise

$$Y_1 = \Delta W_{BP} \quad (A5b)$$

$$U_1 = [\Delta W_w \quad \Delta U_{BP}]^T \quad (A5c)$$

$$A_1 = \begin{bmatrix} -C_{22} & -C_{12} & 0 & 0 \\ 1 & 0 & 0 & 0 \\ 0 & 0 & -C_{21} & -C_{11} \\ 0 & 0 & 1 & 0 \end{bmatrix} \quad (A5d)$$

$$B_1 = [B_{11} \quad B_{21}] \quad (A5e)$$

$$B_{11} = [1 \quad 0 \quad 0 \quad 0]^T$$

$$B_{21} = [0 \quad 0 \quad 1 \quad 0]^T$$

$$C_1 = [d_{22} \quad d_{12} \quad 0 \quad d_{11}] \quad (A5f)$$

#### State-Space Expression of Block II of Fig. 3c

Applying the principle of linear superposition to Fig. 2b, we have the expression

$$\begin{bmatrix} \Delta W_{EN} \\ \Delta N \end{bmatrix} = \begin{bmatrix} h_{11} & h_{12} \\ h_{21} & h_{22} \end{bmatrix} \begin{bmatrix} \Delta W_{BP} \\ \Delta U_F \end{bmatrix} \quad (A6)$$

where

$$h_{11} = K_N G_{IN,1} G_{EN,1}$$

$$h_{12} = K_F K_N G_{EN,2}$$

$$h_{21} = G_{IN,1} G_{EN,1}$$

$$h_{22} = G_{EN,2}$$

Equation (A6) can be transferred into

$$\dot{X}_2 = A_2 X_2 + B_2 U_2 \quad (A7a)$$

$$Y_2 = C_2 X_2 \quad (A7b)$$

where

$$X_2 = [X_5 \quad X_6 \quad X_7 \quad \dots \quad X_{11} \quad X_{12}]^T$$

$$U_2 = [\Delta W_{BP} \quad \Delta W_F]^T, \quad Y_2 = [\Delta W_{EN} \quad \Delta N]^T$$

$$A_2 = \begin{bmatrix} A_{12} & 0 \\ 0 & A_{22} \end{bmatrix}, \quad B_2 = [B_{12} \quad B_{22}], \quad C_2 = \begin{bmatrix} C_{12} \\ C_{22} \end{bmatrix}$$

$$A_{12} = \begin{bmatrix} -\mu_{65} & -\mu_{55} & -\mu_{45} & -\mu_{35} & -\mu_{25} & -\mu_{15} \\ 1 & & & & & \\ & 1 & & & & 0 \\ & & 1 & & & \\ & 0 & & 1 & & \\ & & & & 1 & 0 \end{bmatrix}$$

$$A_{22} = \begin{bmatrix} -\mu_{26} & -\mu_{16} \\ 1 & 0 \end{bmatrix}$$

$$B_{12} = [1 \quad 0 \quad 0 \quad \dots \quad 0]_{8 \times 1}^T$$

$$B_{22} = [0 \quad 0 \quad 0 \quad \dots \quad 1]_{8 \times 1}^T$$

$$C_{12} = [0 \quad \mu_{51} \quad \mu_{41} \quad \mu_{31} \quad \mu_{21} \quad \mu_{11} \quad \mu_{22} \quad \mu_{12}]_{8 \times 1}$$

$$C_{22} = [0 \quad \mu_{53} \quad \mu_{43} \quad \mu_{33} \quad \mu_{23} \quad \mu_{13} \quad \mu_{24} \quad \mu_{14}]_{8 \times 1} \quad (A8)$$

where  $\mu_{65}, \mu_{55}, \dots$  are elements of the matrices  $A_{12}, A_{22}, B_{12}, B_{22}, C_{12}, C_{22}$  [functions of coefficients  $a_{11}, a_{12}, \dots, a_{61}, a_{51}, \dots$  of Eqs. (1) and (3)].

#### State Space Expression of Block III of Fig. 3c

Equation (2) can be transferred into

$$\dot{X}_3 = A_3 X_3 + B_3 U_3 \quad (A9a)$$

$$Y_3 = C_3 X_3 \quad (A9b)$$

where

$$X_3 = [X_{13} \quad X_{14} \quad X_{15} \quad X_{16} \quad X_{17} \quad X_{18}]^T$$

$$Y_3 = \Delta X_S$$

$$U_3 = \Delta W_{JN}$$

$$A_3 = \begin{bmatrix} -a_{62} & -a_{52} & -a_{42} & -a_{32} & -a_{22} & -a_{12} \\ 1 & & & & & \\ & 1 & & & & 0 \\ & & 1 & & & \\ & 0 & & 1 & & \\ & & & & 1 & \\ & & & & & 1 \end{bmatrix}$$

$$B_3 = [1 \quad 0 \quad 0 \quad 0 \quad 0 \quad 0]^T$$

$$C_3 = [0 \quad 0 \quad 0 \quad 0 \quad 0 \quad b_{12}] \quad (A10)$$

#### State-Space Expression of the Inlet/Engine Combination

Combining Eqs. (A4), (A7), and (A9) and also considering Eq. (A10), the required Eq. (11) of the state-space expression of the combination can be obtained.

#### Appendix B: Derivation of Recursive Method of Eigenvalue-Eigenvector

For a linear time-invariant system,

$$\dot{X} = AX + BU \quad (A11a)$$

$$Y = CX \quad (A11b)$$

If the case of general eigenvectors of  $A$  [i.e., suppose  $A$  consists of  $n$  (order of  $A$ ) independent eigenvectors] is not considered, then the following solution can be used.

#### Solution of State Transition Matrix

When the initial state vector is  $X_0$ , the solution of Eq. (A11) will be

$$Y(t) = C \left\{ e^{At} X_0 + \int_0^t e^{A(t-\tau)} B U(\tau) d\tau \right\} \quad (A12)$$

Let  $\lambda_1, \dots, \lambda_p, \lambda_{i+1}, \dots, \lambda_n$  and  $r_1, r_2, \dots, r_p, r_{i+1}, \dots, r_n$  are eigenvalues and eigenvectors of matrix  $A$ , respectively. While  $\lambda_i$  and  $\lambda_{i+1}$  are a pair of corresponding conjugate eigenvectors.

In order to avoid computation of complex numbers and, thus, to save computer time, the real and imaginary parts of the

complex conjugate eigenvectors of  $\mathbf{A}$  can be written separately as

$$\mathbf{r}_i = \mathbf{r}_{rei} + j\mathbf{r}_{lmi} \quad (\text{A13a})$$

$$\mathbf{r}_{i+1} = \mathbf{r}_{rei} - j\mathbf{r}_{lmi} \quad (\text{A13b})$$

and let

$$\mathbf{r}'_i = \mathbf{r}_{rei} + \mathbf{r}_{lmi} \quad (\text{A14a})$$

$$\mathbf{r}'_{i+1} = \mathbf{r}_{rei} - \mathbf{r}_{lmi} \quad (\text{A14b})$$

then

$$\mathbf{A} = [\mathbf{r}'_i \quad \mathbf{r}'_{i+1}] = [\mathbf{r}_i \quad \mathbf{r}_{i+1}] \begin{bmatrix} \theta_i & -\eta_i \\ \eta_i & \theta_i \end{bmatrix}$$

where  $\theta_i$  and  $\eta_i$  are determined from

$$\lambda_i = \theta_i + j\eta_i \quad (\text{A15a})$$

$$\lambda_{i+1} = \theta_i - j\eta_i \quad (\text{A15b})$$

Substitute for the  $i$ th and  $(i+1)$ st column,  $\mathbf{r}_i$  and  $\mathbf{r}_{i+1}$ , of the eigenvector matrix  $\mathbf{T}_v$  with  $\mathbf{r}'_i$  and  $\mathbf{r}'_{i+1}$ , respectively, and all other complex conjugate eigenvectors of  $\mathbf{T}_v$  are substituted in the same way. Thus, a new real eigenvector matrix  $\mathbf{T}'_v$  is formulated. Take  $\mathbf{T}'_v$  as a linear transformation matrix of

$$\mathbf{X} = \mathbf{T}'_v \mathbf{X}'$$

and Eqs. (A11) can be transformed into

$$\dot{\mathbf{X}}' = \mathbf{A}' \mathbf{X}' + \mathbf{B}' \mathbf{U} \quad (\text{A16a})$$

$$\mathbf{Y} = \mathbf{C}' \mathbf{X}' \quad (\text{A16b})$$

where

$$\mathbf{A}' = (\mathbf{T}'_v)^{-1} \mathbf{A} \mathbf{T}'_v$$

$$\mathbf{B}' = (\mathbf{T}'_v)^{-1} \mathbf{B}$$

$$\mathbf{C}' = \mathbf{C} \mathbf{T}'_v$$

$$\mathbf{A}' = \begin{bmatrix} \lambda_1 & & & & \\ & \ddots & & & \\ & & \lambda_{i-1} & & \\ & & & \begin{bmatrix} \theta_i & -\eta_i \\ \eta_i & \theta_i \end{bmatrix} & \\ & & & & \lambda_{i+1} \\ & & & & & \ddots \\ & & & & & & \lambda_n \end{bmatrix} \quad (\text{A17})$$

From Eqs. (A12) and (A17), solution  $\mathbf{Y}$  can be rewritten as

$$\mathbf{Y} = \mathbf{C}' \left[ e^{\mathbf{A}'t} \mathbf{X}'_0 + \int_0^t e^{\mathbf{A}'(t-\tau)} \mathbf{B}' \mathbf{U}(\tau) d\tau \right] \quad (\text{A18})$$

where

$$\mathbf{X}'_0 = (\mathbf{T}'_v)^{-1} \mathbf{X}_0 \quad (\text{A19})$$

$$e^{\mathbf{A}'t} = \begin{bmatrix} e^{\lambda_1 t} & & & & \\ & \ddots & & & \\ & & e^{\lambda_{i-1} t} & & \\ & & & e^{\begin{bmatrix} \theta_i & -\eta_i \\ \eta_i & \theta_i \end{bmatrix} t} & \\ & & & & e^{\lambda_{i+1} t} \\ & & & & & \ddots \\ & & & & & & e^{\lambda_n t} \end{bmatrix} \quad (\text{A20})$$

and

$$e^{\begin{bmatrix} \theta_i & -\eta_i \\ \eta_i & \theta_i \end{bmatrix} t} = \begin{bmatrix} e^{\theta_i t} \cos(\eta_i t) & -e^{\theta_i t} \sin(\eta_i t) \\ e^{\theta_i t} \sin(\eta_i t) & e^{\theta_i t} \cos(\eta_i t) \end{bmatrix} \quad (\text{A21})$$

#### Recursive Computation Formulas

In order to save computer time further, a recursive computation formula of  $\mathbf{Y}(t)$  is deduced to obtain direct numerical integration of  $\mathbf{Y}(t)$  from Eq. (A18).

Suppose  $\mathbf{Y}(t_i)$  at time  $t_i$  is known. Let

$$\mathbf{Y}(t_i) = \mathbf{C}' \Phi(t_i) \quad (\text{A22a})$$

where

$$\Phi(t_i) = e^{\mathbf{A}'t_i} \mathbf{X}'_0 + \int_0^{t_i} e^{\mathbf{A}'(t_i-\tau)} \mathbf{B}' \mathbf{U}(\tau) d\tau \quad (\text{A22b})$$

Let

$$t_{i+1} = t_i + dt_i$$

where  $dt_i$  is the time step. Then, from Eq. (A22)

$$\mathbf{Y}(t_{i+1}) = \mathbf{C}' \Phi(t_{i+1}) \quad (\text{A23a})$$

where

$$\Phi(t_{i+1}) = e^{\mathbf{A}'dt_i} \Phi(t_i) + \int_{t_i}^{t_i+dt_i} e^{\mathbf{A}'(t_i+dt_i-\tau)} \mathbf{B}' \mathbf{U}(\tau) d\tau \quad (\text{A23b})$$

By the transformation of

$$v = \tau - t_i \quad (\text{A24})$$

We have

$$dv = d\tau \quad (\text{A25})$$

where  $t_i$  is the initial value.

Substituting Eqs. (A24) and (A25) into Eq. (A23), we have

$$\Phi(t_{i+1}) = e^{\mathbf{A}'dt_i} \Phi(t_i) + \int_0^{dt_i} e^{\mathbf{A}'(dt_i-v)} \mathbf{B}' \mathbf{U}(t_i+v) dv \quad (\text{A26})$$

From Eq. (A22), initial value of  $\Phi(0)$  is determined by

$$\Phi(0) = e^{\mathbf{A}'0} \mathbf{X}'_0 + \int_0^0 e^{\mathbf{A}'(0-\tau)} \mathbf{B}' \mathbf{U}(\tau) d\tau = \mathbf{X}'_0 \quad (\text{A27})$$

In general, the control vector  $\mathbf{U}(\tau)$  satisfies the condition

$$\mathbf{U}(\tau) = \mathbf{U}(t_i) = \text{const vector } (t_i \leq \tau \leq t_{i+1}) \quad (\text{A28})$$

Then, Eq. (A22) becomes

$$\begin{aligned} \Phi(t_{i+1}) &= e^{\mathbf{A}'dt_i} \Phi(t_i) + \int_0^{dt_i} e^{\mathbf{A}'(dt_i-v)} \mathbf{B}' dv \mathbf{U}(t_i) \\ &= \mathbf{P}_{cm}(dt_i) \Phi(t_i) + \mathbf{H}_{cm}(dt_i) \mathbf{U}(t_i) \end{aligned} \quad (\text{A29})$$



where

$$\mathbf{P}_{cm}(dt_i) = e^{\mathbf{A}' dt_i} \quad (\text{A30a})$$

$$\mathbf{H}_{cm}(dt_i) = \int_0^{dt_i} e^{\mathbf{A}'(dt_i - v)} \mathbf{B}' dv \quad (\text{A30b})$$

Apparently,  $\mathbf{P}_{cm}$  and  $\mathbf{H}_{cm}$  depend upon time step  $dt_i$  only.

Combining Eqs. (A22) and (A27) to (A30), we have the recursive computation equations (24) and (25).

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